# Steam Condensation Waves in Water-Saturated Porous Rock<sup>1</sup>

Johannes Bruining

Dietz Laboratory, Centre of Technical Geoscience Mijnbouwstraat 120 2628 RX Delft, The Netherlands E-mail: J.Bruining@ta.tudelft.nl

Dan Marchesin

Instituto Nacional de Matemática Pura e Aplicada Estrada Dona Castorina 110 22460-320 Rio de Janeiro, RJ, Brazil E-mail: marchesi@impa.br

Stephen Schecter

Mathematics Department North Carolina State University Raleigh, NC 27695, USA E-mail: schecter@math.ncsu.edu

Submitted: August 6, 2003 Accepted: January 25, 2004

Dedicated to Jorge Sotomayor on his 60th birthday

We formulate balance laws governing condensation of steam injected into a linear porous medium containing water. Heat losses to the outside are neglected. Longitudinal heat conduction and capillary effects are taken into account. The condensation process is modeled by a rate equation based on a simple heat-transfer model. We study the condensation front as a traveling wave, under the approximation that pressure variations are negligible within the front. We find this traveling wave for the case of injection of high-quality steam using a combination of phase-plane analysis and numerical calculation of orbits.

Key Words: Porous medium, steamflood, traveling waves, multiphase flow

<sup>1</sup>This work was supported in part by: CNPq under Grant 301532/2003-6, FINEP under CTPETRO Grant 21.01.0248.00; NWO under Fellowship Grant R 75-389, The Netherlands; NSF under Grant DMS-9973105; IMA (Univ. of Minnesota), Dietz Laboratory (TU Delft, The Netherlands), IMPA (Brazil).

### 1. INTRODUCTION

Steam injection is an effective technique to restore groundwater aquifers contaminated with non-aqueous-phase liquids (NAPL's) such as hydrocarbon fuels and halogenated hydrocarbons [17]. It is also one of the most effective methods to recover oil from medium to heavy oil reservoirs [16]. The main feature of steam injection is the steam condensation front (SCF), which marks the boundary between the upstream zone at boiling temperature and the downstream liquid zone below the boiling temperature. The main result of this work is the analysis of the structure of the SCF in the absence of NAPL's.

There is an abundant engineering literature on numerical ([17], [20]), experimental ([9]), and theoretical ([1], [8], [11], [12], [13], [18], [19], [21], [22], [24], [25], [26]) modeling of steam injection for clean-up and oil recovery. The main focus of the experimental and theoretical work is the internal structure of the steam condensation front. This literature is reviewed in [5].

In the latter work, a classification of Riemann solutions for steam injection in water is also presented. However, the fact that the steam condensation front is the small "diffusion" limit of a traveling wave is assumed in [5], rather than proved. The purpose of the present work is to show that this assumption is correct for the particular case where the steam condensation front arises in the injection of steam with little water. In this case, the SCF ceases to obey Lax's entropy condition at the injection end. In other words, in the associated ODE, the left equilibrium has the extra technical difficulty of being non-hyperbolic.

In Section 2, the physical model is summarized. (The interested reader can find a detailed description in [5].) It is described mathematically by balance equations of mass and thermal energy, which are rewritten into a form suitable for analysis.

In Section 3, we derive the ordinary differential equation boundary value problem for the traveling wave that represents the *SCF*. In Section 4, some useful facts about the traveling wave ODE are proved. In Section 5, we show that existence of a solution to the ODE boundary value problem is plausible. Section 6 proves the existence of a solution; however, one fact needed in the proof is verified numerically rather than analytically. Appendix A gives notation and values for the physical quantities appearing in the model. In Appendix A.2 we calculate the speed of the steam condensation front.

#### 2. PHYSICAL AND MATHEMATICAL MODEL

#### 2.1. Physical model

We consider linear steam displacement in a homogeneous reservoir of constant permeability and porosity. The reservoir is initially saturated with water. The pressure gradients  $\partial p_w / \partial x$ ,  $\partial p_g / \partial x$  driving the fluids are small with respect to the prevailing system pressure p divided by the length of the reservoir. In particular, within the short steam condensation zone pressure variations are negligible. Hence we disregard the effect of pressure variation on the density of the fluids and on their thermodynamic properties. The reservoir is horizontal, so gravitational effects can be ignored. A steam-water mixture is injected at constant rate and constant composition. Transverse heat losses are disregarded.

The relevant effects of temperature on the fluid properties, *e.g.*, steam density  $\rho_g$ , the water density  $\rho_w$ , water viscosity  $\mu_w$ , and steam viscosity  $\mu_g$  are taken into account (see also Appendix A). Darcy's Law determines the fluid motion. The (non-linear) temperature dependence of enthalpies and of the evaporation heat are also taken into account (see also Appendix A). Capillary pressure as well as an effective longitudinal heat conduction term are included.

We have chosen to describe condensation in terms of a steam mass condensation rate equation. The mass condensation rate q is always positive when the temperature drops below the boiling temperature  $T^b$  as long as not all steam has condensed, that is,  $S_w < 1$ .

The stated conditions can be considered representative of steam injection in the subsurface for remediation of contaminated sites. As steam is injected the reservoir is heated. Depending on the proportions of steam and water in the injected mixture, we can distinguish three regimes, which differ in the structure of Riemann solutions. In two of the three regimes, there is an SCF. To the right of the SCF, there is pure water.

Each of the enthalpies per unit mass  $h_w(T)$ ,  $h_r(T)$ ,  $h_g(T)$  [J/kg] is defined with respect to the enthalpy at the reference temperature  $T^0$ . In particular,

$$h_w(T^0) = h_r(T^0) = 0. (1)$$

The enthalpy of steam is subdivided into a sensible part (called sensible heat)  $h_g^s(T)$  and a latent part (called latent heat)  $h_g^l(T^0)$ , *i.e.*,  $h_g(T) = h_a^s(T) + h_a^l(T^0)$ . We have  $h_a^s(T^0) = 0$  and

$$h_q^l(T^0) = \bar{\Lambda},\tag{2}$$

where  $\bar{\Lambda}$  is the evaporation heat or the latent heat per unit mass.

The enthalpies per unit volume are  $H_w(T) = \rho_w(T)h_w(T)$ ,  $H_r(T) = \rho_r(T)h_r(T)$ ,  $H_g^s(T) = \rho_g(T)h_g^s(T)$   $[J/m^3]$ , which vanish at  $T^0$ , the initial

reservoir temperature, and

$$H_q^l(T) = \rho_g(T)\bar{\Lambda}.$$
(3)

The enthalpies as functions of temperature are summarized in Appendix A for convenience.

We assume Darcy's law without gravity for the flow rates  $u_w$ ,  $u_g$  of water and steam:

$$u_w = -\frac{kk_{rw}}{\mu_w}\frac{\partial p_w}{\partial x}, \qquad u_g = -\frac{kk_{rg}}{\mu_g}\frac{\partial p_g}{\partial x}.$$
 (4)

Expressions for the relative permeabilities  $k_{rw}$ ,  $k_{rg}$  are given in Eqs. A.10 and A.11, and the difference  $p_g - p_w = P_c$  (also called the capillary pressure) is given by Eq. A.12.

We use the Brooks-Corey power law dependence of the capillary pressure on saturation (see Appendix A).

The water mass source term is taken as

$$q = \begin{cases} q_b(T - T^b)(S_w - 1) & \text{if } T \le T^b \text{ and } 0 \le S_w \le 1, \\ 0 & \text{otherwise.} \end{cases}$$
(5)

This term is motivated by the idea that the condensation rate is determined by a "driving force" that is proportional to the departure of temperature from from equilibrium  $T^b - T$  (see also [15]) and the steam saturation  $(S_g)$ . The value of  $q_b$  is considered to be very large.

#### 2.2. The model equations

The mass balance equations of liquid water and steam read as follows:

$$\frac{\partial(\varphi\rho_w S_w)}{\partial t} + \frac{\partial(\rho_w u_w)}{\partial x} = q, \qquad (6)$$

$$\frac{\partial(\varphi\rho_g S_g)}{\partial t} + \frac{\partial(\rho_g u_g)}{\partial x} = -q.$$
(7)

The rock porosity  $\varphi$  is assumed to be constant. The densities of pure liquid water and pure steam are  $\rho_w$  and  $\rho_g$ . In our derivations below we ignore the small temperature dependence of the water density. The steam density depends on both pressure and temperature, but we disregard the small pressure effects due to the Darcy flow of the fluids on the density and all other thermodynamic quantities. We include longitudinal heat conduction, but neglect heat losses to the surrounding rock, in the energy balance equation given below. By our assumption of almost constant pressure we

ignore adiabatic compression and decompression effects. Thus the energy balance is (see [3], Table 10.4-1)

$$\frac{\partial}{\partial t}(H_r + \varphi S_w H_w + \varphi S_g H_g) + \frac{\partial}{\partial x}(u_w H_w + u_g H_g) = \frac{\partial}{\partial x}\left(\kappa \frac{\partial T}{\partial x}\right).$$
 (8)

Here  $\kappa$  is the composite conductivity of the rock–fluid system [2], given by an equation such as

$$\kappa = \kappa_r + \varphi(S_w \kappa_w + S_g \kappa_g). \tag{9}$$

It turns out that  $\kappa > \kappa_r > 0$  is all we need from the formula (9). Equations (6), (7), and (8) are the basic equations.

Equations (6) and (7) are combined with the heat balance equation (8), where we also use separation in sensible and latent quantities, to obtain:

$$\frac{\partial}{\partial t} \left( H_r + \varphi S_w H_w + \varphi S_g H_g^s \right) + \frac{\partial}{\partial x} \left( u_w H_w + u_g H_g^s \right) - \frac{\partial}{\partial x} \left( \kappa \frac{\partial T}{\partial x} \right) =$$
$$= -\frac{\partial}{\partial t} \left( \varphi S_g H_g^l \right) - \frac{\partial}{\partial x} \left( u_g H_g^l \right) = -\frac{\partial}{\partial t} \left( \varphi S_g \rho_g \bar{\Lambda} \right) - \frac{\partial}{\partial x} \left( u_g \rho_g \bar{\Lambda} \right) =$$
$$= -\bar{\Lambda} \left( \frac{\partial}{\partial t} \left( \varphi S_g \rho_g \right) + \frac{\partial}{\partial x} \left( u_g \rho_g \right) \right).$$

Using Eq. (7), this yields

$$\frac{\partial}{\partial t} \left( H_r + \varphi S_w H_w + \varphi S_g H_g^s \right) + \frac{\partial}{\partial x} \left( u_w H_w + u_g H_g^s \right) = = q \bar{\Lambda} + \frac{\partial}{\partial x} \left( \kappa \frac{\partial T}{\partial x} \right).$$
(10)

Let us define the fractional flow functions for water and steam:

$$f_w = \frac{k_{rw}/\mu_w}{k_{rw}/\mu_w + k_{rg}/\mu_g}, \qquad f_g = \frac{k_{rg}/\mu_g}{k_{rw}/\mu_w + k_{rg}/\mu_g}.$$
 (11)

The capillary pressure

$$P_c = P_c(S_w) = p_g - p_w,$$
 (12)

which is given by Equation (A.12), is a strictly monotone decreasing function; it appears in the definition of the capillary diffusion coefficient

$$\Omega = -f_w \frac{kk_{rg}}{\mu_g} \frac{dP_c}{dS_w} \ge 0.$$
(13)

We notice that  $\Omega$  vanishes precisely at water saturations  $S_w = S_{wc}$  and  $S_w = 1$  ( $f_w(S_w) = 0$  for  $S_w \leq S_{wc}$ ). We will invoke the presence of diffusion due to physical effects other than capillary pressure to add a small positive quantity to  $\Omega$  so it does not vanish near  $S_w = 1$ .

Using Darcy's law (4) and the definition of  $P_c$  given in Eq. (12), one can easily show from Eqs. (4) and (13) that

$$u_w = u f_w - \Omega \frac{\partial S_w}{\partial x}, \qquad u_g = u f_g - \Omega \frac{\partial S_g}{\partial x},$$
 (14)

where

$$u = u_w + u_g \tag{15}$$

is the *total* or *Darcy velocity* and  $\Omega$  acts as a saturation-dependent capillary diffusion coefficient.

Substituting (14) into Equations (6), (7) and (10) leads to

$$\varphi \frac{\partial \left(\rho_w S_w\right)}{\partial t} + \frac{\partial \left(\rho_w u f_w\right)}{\partial x} = q + \frac{\partial}{\partial x} \left(\rho_w \Omega \frac{\partial S_w}{\partial x}\right), \tag{16}$$

$$\varphi \frac{\partial \left(\rho_g S_g\right)}{\partial t} + \frac{\partial \left(\rho_g u f_g\right)}{\partial x} = -q + \frac{\partial}{\partial x} \left(\rho_g \Omega \frac{\partial S_g}{\partial x}\right),\tag{17}$$

$$\frac{\partial}{\partial t} \left( H_r + \varphi H_w S_w + \varphi H_g^s S_g \right) + \frac{\partial}{\partial x} \left( u \left( H_w f_w + H_g^s f_g \right) \right) = 
= q \bar{\Lambda} + \frac{\partial}{\partial x} \left( \left( H_w - H_g^s \right) \Omega \frac{\partial S_w}{\partial x} \right) + \frac{\partial}{\partial x} \left( \kappa \frac{\partial T}{\partial x} \right),$$
(18)

where on the right-hand side of Eq. (18) we used that  $\partial S_w/\partial x = -\partial S_g/\partial x$ . The governing system of equations can be rewritten as (16)–(18).

# 3. BLOW-UP OF THE STEAM CONDENSATION ZONE

In this section we assume that time and distance have been scaled so that the diffusion terms and the heat condensation terms balance. Therefore the steam condensation zone has width O(1); it will be studied as a traveling wave extending from  $-\infty$  to  $+\infty$ . In the steam condensation zone capillary pressure effects, conduction, mass and heat transfer terms all have to be taken into account. However, because this zone is thin relative to the reservoir, we are always justified in ignoring pressure variation effects on densities within the zone.

For the analysis of traveling waves, it is advantageous to define the cumulative condensation distribution Q(x,t) and cumulative condensation

 $Q^+(t)$  by

$$Q(x,t) = \int_{-\infty}^{x} q(x',t)dx', \qquad Q^{+}(t) = \int_{-\infty}^{+\infty} q(x',t)dx'.$$
(19)

We look for solutions of Eqs. (19) and (16)–(18) that depend only on  $\xi = (x - v^{SCF})t$ , where  $v^{SCF}$  is the speed of the steam condensation traveling wave. Such a solution must satisfy the system

$$\frac{dQ}{d\xi} = q, \tag{20}$$

$$-\varphi v^{SCF} \frac{d\left(\rho_w S_w\right)}{d\xi} + \frac{d\left(\rho_w u f_w\right)}{d\xi} = \frac{dQ}{d\xi} + \frac{d}{d\xi} \left(\rho_w \Omega \frac{dS_w}{d\xi}\right), \quad (21)$$

$$-\varphi v^{SCF} \frac{d\left(\rho_g S_g\right)}{d\xi} + \frac{d\left(\rho_g u f_g\right)}{d\xi} = -\frac{dQ}{d\xi} + \frac{d}{d\xi} \left(\rho_g \Omega \frac{dS_g}{d\xi}\right), \quad (22)$$

$$-v^{SCF}\frac{d}{d\xi}\left(H_r + \varphi H_w S_w + \varphi H_g^s S_g\right) + \frac{d}{d\xi}\left(u\left(H_w f_w + H_g^s f_g\right)\right) = = \bar{\Lambda}\frac{dQ}{d\xi} + \frac{d}{d\xi}\left(\left(H_w - H_g^s\right)\Omega\frac{dS_w}{d\xi}\right) + \frac{d}{d\xi}\left(\kappa\frac{dT}{d\xi}\right).$$
(23)

We look for solutions  $(Q(\xi),S_w(\xi),S_g(\xi),T(\xi))$  of Eqs. (20)–(23) that join two equilibria. More precisely

$$\lim_{\xi \to -\infty} (Q(\xi), S_w(\xi), S_g(\xi), T(\xi)) = (Q^-, S_w^-, S_g^-, T^-),$$
$$\lim_{\xi \to \infty} (Q(\xi), S_w(\xi), S_g(\xi), T(\xi)) = (Q^+, S_w^+, S_g^+, T^+),$$
$$\lim_{\xi \to \pm \infty} \left( \frac{dQ}{d\xi}, \frac{dS_w}{d\xi}, \frac{dS_g}{d\xi}, \frac{dT}{d\xi} \right) = 0,$$

with  $Q^- = 0$ ,  $S_w^-$  and  $u^-$  given,  $S_g^- = 1 - S_w^-$ ,  $T^- = T^b$ ,  $Q^+$  to be determined,  $S_w^+ = 1$ ,  $S_g^+ = 0$ ,  $T^+ = T^0$ . Since steam is injected at the left, we assume  $u^- > 0$ .

We integrate Eqs. (21)–(23) from  $-\infty$  to  $\xi$ , and use Eq. (5), obtaining

$$\frac{dQ}{d\xi} = \begin{cases} q_b(T - T^b)(S_w - 1) & \text{for } T \le T^b, & 0 \le S_w \le 1; \\ 0 & \text{otherwise.} \end{cases}$$
(24)

$$\rho_w \Omega \frac{dS_w}{d\xi} = (-Q - \varphi v^{SCF} \rho_w S_w + \rho_w u f_w) - (-\varphi v^{SCF} \rho_w^- S_w^- + \rho_w^- u^- f_w^-),$$

$$(25)$$

$$-\rho_g \Omega \frac{dS_g}{d\xi} = (-Q + \varphi v^{SCF} \rho_g S_g - \rho_g u f_g) - (\varphi v^{SCF} \rho_g^- S_g^- - \rho_g^- u^- f_g^-),$$

$$(26)$$

$$(H_w - H_g^s)\Omega \frac{dS_w}{d\xi} + \kappa \frac{dT}{d\xi} = -v^{SCF}(H_r + \varphi H_w S_w + \varphi H_g^s S_g) + u(H_w f_w + H_g^s f_g) - \bar{\Lambda}Q + v^{SCF}(H_r^- + \varphi H_w^- S_w^- + \varphi H_g^{s-} S_g^-) - u^-(H_w^- f_w^- + H_g^{s-} f_g^-).$$
(27)

To take advantage of proportionality between certain quantities, we introduce

$$\widetilde{u} = \frac{u}{u^{-}}, \quad \widetilde{v}^{SCF} = \frac{\varphi v^{SCF}}{u^{-}}, \quad \widetilde{u}^{+} = \frac{u^{+}}{u^{-}}, \quad \widetilde{Q}^{+} = \frac{Q^{+}}{u^{-}},$$

$$\widetilde{q}_{b} = \frac{q_{b}}{u^{-}}, \quad \widetilde{\Omega} = \frac{\Omega}{u^{-}}, \quad \widetilde{\kappa} = \frac{\kappa}{u^{-}}.$$
(28)

Since  $S_w^+ = 1$  and  $S_g^+ = 0$ , we see that  $f_w^+ = 1$  and  $f_g^+ = 0$ . By letting  $\xi$  tend to  $+\infty$  in Eq. (25), we obtain

$$\widetilde{Q}^{+} = \widetilde{u}^{+} \rho_{w}^{+} - \rho_{w}^{-} f_{w}^{-} - \widetilde{v}^{SCF} (\rho_{w}^{+} - \rho_{w}^{-} S_{w}^{-}).$$
<sup>(29)</sup>

Similarly, Eq. (26) yields

$$\widetilde{Q}^{+} = \rho_{g}^{-} \left( f_{g}^{-} - \widetilde{v}^{SCF} S_{g}^{-} \right).$$
(30)

Since  $T^+ = T^0$ , we use Eq. (1) in Eq. (27) to obtain the Rankine–Hugoniot condition

$$\bar{\Lambda}\widetilde{Q}^{+} = -(H_{w}^{-}f_{w}^{-} + H_{g}^{s-}f_{g}^{-}) + \widetilde{v}^{SCF}\left(H_{r}^{-}/\varphi + H_{w}^{-}S_{w}^{-} + H_{g}^{s-}S_{g}^{-}\right).$$
 (31)

Eqs. (29), (30) and (31) are a linear system in the unknowns  $\tilde{v}^{SCF}$ ,  $\tilde{u}^+$  and  $\tilde{Q}^+$ . Since  $S_w + S_g = 1$ , one of the Eqs. (25)–(27) can be replaced by an equation for the total velocity  $\tilde{u}$ . To eliminate one of the Eqs. (25)–(27), we divide Eqs. (25), (26) by the densities, and obtain in the new notation

$$(28)$$
:

$$\widetilde{\Omega}\frac{dS_w}{d\xi} = \left(-\frac{\widetilde{Q}}{\rho_w} - \widetilde{v}^{SCF}S_w + \widetilde{u}f_w\right) - \frac{\rho_w^-}{\rho_w}\left(-\widetilde{v}^{SCF}S_w^- + f_w^-\right), \quad (32)$$

$$-\widetilde{\Omega}\frac{dS_g}{d\xi} = \left(-\frac{\widetilde{Q}}{\rho_g} + \widetilde{v}^{SCF}S_g - \widetilde{u}f_g\right) - \frac{\rho_g^-}{\rho_g}\left(\widetilde{v}^{SCF}S_g^- - f_g^-\right).$$
(33)

Since  $S_w + S_g = 1$ ,  $\frac{dS_w}{d\xi} = -\frac{dS_g}{d\xi}$ . Therefore, subtracting Eq. (33) from Eq. (32) we obtain  $\tilde{u} = \tilde{u}(\tilde{Q}, T)$  as

$$\widetilde{u} = \widetilde{Q} \left( \frac{1}{\rho_w} - \frac{1}{\rho_g} \right) + \widetilde{v}^{SCF} + \frac{\rho_w^-}{\rho_w} \left( -\widetilde{v}^{SCF} S_w^- + f_w^- \right) + \frac{\rho_g^-}{\rho_g} \left( -\widetilde{v}^{SCF} S_g^- + f_g^- \right).$$
(34)

Remark 1. Letting  $\xi \to +\infty$  in Eq. (34), we obtain  $\widetilde{u}^+$  as

$$\widetilde{u}^{+} = \widetilde{Q}^{+} \left( \frac{1}{\rho_{w}^{+}} - \frac{1}{\rho_{w}^{-}} \right) + \widetilde{v}^{SCF} +$$

$$+ \left( \rho_{w}^{-} h_{w}^{+} - \rho_{g}^{-} h_{g}^{+} \right) \left( f_{w}^{-} - \widetilde{v}^{SCF} S_{w}^{-} \right) + \rho_{g}^{-} h_{g}^{+} \left( 1 - \widetilde{v}^{SCF} \right).$$

$$(35)$$

This equation can also be obtained from Eqs. (29) and (30).

Multiplying Eq. (32) by  $H_w$ , Eq. (33) by  $H_g^s$ , and subtracting we obtain

Subtracting this equation from Eq. (27) divided by  $u^-$  we obtain

$$\widetilde{\kappa} \frac{\partial T}{\partial \xi} = \widetilde{Q} \left( h_w - h_g^s - \overline{\Lambda} \right) - \\
- \widetilde{v}^{SCF} \left( \rho_w^- \left( h_w - h_w^- \right) S_w^- + \rho_g^- \left( h_g^s - h_g^{s-} \right) S_g^- \right) - \\
- \widetilde{v}^{SCF} (H_r - H_r^-) / \varphi + \left( \rho_w^- \left( h_w - h_w^- \right) f_w^- + \rho_g^- \left( h_g^s - h_g^{s-} \right) f_g^- \right).$$
(37)

Subtracting Eq. (25) from Eq. (26) we obtain

$$(\rho_w - \rho_g)\widetilde{\Omega}\frac{dS_w}{d\xi} = -\widetilde{v}^{SCF}(\rho_w S_w + \rho_g S_g) + \widetilde{u}(\rho_w f_w + \rho_g f_g) + \widetilde{v}^{SCF}(\rho_w^- S_w^- + \rho_g^- S_g^-) - (\rho_w^- f_w^- + \rho_g^- f_g^-).$$
(38)

The system of ODE's that we shall study consists of Eqs. (24), (37) and (38), in the independent variables  $(\tilde{Q}, T, S_w)$ . To put these equations in final form, we divide Eq. (37) by  $\tilde{\kappa}$  and Eq. (38) by  $(\rho_w - \rho_g) \tilde{\Omega}$ , and in Eq. (38) we substitute Eq. (34) for  $\tilde{u}$ . We obtain

$$\frac{d\widetilde{Q}}{d\xi} = \begin{cases} q_b(T - T^b)(S_w - 1) & \text{if } T \le T^b \text{ and } 0 \le S_w \le 1, \\ 0 & \text{otherwise,} \end{cases}$$
(39)

$$\frac{dT}{d\xi} = \widetilde{Q}A(T) + B(T), \qquad (40)$$

$$\frac{dS_w}{d\xi} = \widetilde{Q}C(T, S_w) + D(T, S_w), \qquad (41)$$

with

$$\widetilde{\kappa}A(T) = h_w - h_g^s - \overline{\Lambda},$$

$$\widetilde{\kappa}B(T) = -\widetilde{v}^{SCF} \left( \rho_w^-(h_w - h_w^-)S_w^- + \rho_g^-(h_g^s - h_g^{s-})S_g^- \right)$$
(42)

$$-\tilde{v}^{SCF}(H_r - H_r^-)/\varphi + \left(\rho_w^-(h_w - h_w^-)f_w^- + \rho_g^-(h_g^s - h_g^{s-})f_g^-\right), \quad (43)$$

$$(\rho_w - \rho_g) \widetilde{\Omega} C(T, S_w) = \left(\frac{1}{\rho_w} - \frac{1}{\rho_g}\right) (\rho_w f_w + \rho_g f_g), \qquad (44)$$
$$(\rho_w - \rho_g) \widetilde{\Omega} D(T, S_w) = -\widetilde{v}^{SCF} (\rho_w S_w + \rho_g S_w)$$

$$(\rho_w - \rho_g) \Omega D(1, S_w) = -v^{-v} (\rho_w S_w + \rho_g S_g)$$

$$+ \left( \tilde{v}^{SCF} + \frac{\rho_w^-}{\rho_w} \left( -\tilde{v}^{SCF} S_w^- + f_w^- \right) + \frac{\rho_g^-}{\rho_g} \left( -\tilde{v}^{SCF} S_g^- + f_g^- \right) \right) (\rho_w f_w + \rho_g f_g)$$

$$+ \tilde{v}^{SCF} (\rho_w^- S_w^- + \rho_g^- S_g^-) - (\rho_w^- f_w^- + \rho_g^- f_g^-).$$

$$(45)$$

The right side of Eq. (42) has the meaning of minus the heat released at condensation. Physics dictates that this heat is positive, so we assume

$$A(T) < 0 \text{ for } T^0 \le T \le T^b.$$

$$\tag{46}$$

It follows from Eqs. (A.16) and (A.17) that  $\tilde{\kappa}B(T) > 0$ . Physics also dictates that the enthalpies are strictly increasing functions of T. Therefore we have that

$$B(T)$$
 is a strictly decreasing function of T for  $T^0 \le T \le T^b$ . (47)

## 4. PROPERTIES OF THE TRAVELING WAVE ODE

In Eqs. (42)–(45), we recall that  $T^- = T^b$ ,  $S_w^-$  is given, and  $S_g^- = 1 - S_w^-$ . We then define  $\tilde{v}^{SCF}$  by Eq. (A.16).  $\tilde{Q}^+$  is given by Eq. (30).

As explained in [5], the steam condensation front has the following properties when pure or almost pure steam is injected in water:

I. The SCF is preceded by a rarefaction wave adjacent at its left. In other words, it is a *left-characteristic* shock, i.e.

$$\tilde{v}^{SCF} = \frac{\partial f_w}{\partial S_w} (T^-, S_w^-).$$
(48)

II. For the *SCF* between  $S_w^-$  and  $S_w^+$ , the condensation is complete, *i.e.*,  $S_w^+ = 1$  and  $T^+ = T^0$ . (This was noted in the previous section.)

III. One can show based on I, II, and the S-shape of  $f_w$ , which arises from the behavior of  $k_w$  and  $k_g$ , that

$$\left(\frac{\partial^2 f_w}{\partial S_w^2}\right)^- > 0. \tag{49}$$

IV. One can show based on I, II, the S-shape of  $f_w$ , and the dependence of  $\mu_g$  and  $\mu_w$  on temperature, that

$$(f_w - f_w^-) - \tilde{v}^{SCF}(S_w - S_w^-) > 0 \quad \text{for } S_w^- < S_w < 1$$
(50)

(Oleĭnik's inequality), and

$$-\tilde{v}^{SCF}S_g^- + f_g^- > 0. \tag{51}$$

Another fact that we will use quite often is that by the definition of liquid and gas phases,

$$\rho_w - \rho_g > 0$$

away from the critical point.

THEOREM 1. The system (39)-(41) has the following properties:

 $(1)A(T) < 0 \text{ for } T^0 \le T \le T^b.$ 

(2)B(T) is a strictly decreasing function of T for  $T^0 \leq T \leq T^b$ , and  $B(T^b) = 0$ .

 $\begin{aligned} (3)\widetilde{Q}^{+}A(T^{0}) + B(T^{0}) &= 0. \\ (4)\widetilde{Q}^{+}A(T) + B(T) < 0 \ for \ T^{0} < T \le T^{b}. \\ (5)C(T, S_{w}) < 0 \ for \ T^{0} \le T \le T^{b} \ and \ S_{w}^{-} \le S_{w} \le 1. \end{aligned}$ 

$$\begin{split} (6)D(T^{b},S_{w}^{-}) &= 0.\\ (7)\frac{\partial D}{\partial S_{w}}(T^{b},S_{w}^{-}) &= 0.\\ (8)\frac{\partial^{2} D}{\partial S_{w}^{2}}(T^{b},S_{w}^{-}) &> 0.\\ (9)D(T^{b},S_{w}) &> 0 \ for \ S_{w}^{-} < S_{w} \leq 1.\\ (10)\widetilde{Q}^{+}C(T,1) + D(T,1) &= 0 \ for \ T^{0} \leq T \leq T^{b}.\\ (11)\frac{\partial D}{\partial T}(T^{b},S_{w}^{-}) &> 0. \end{split}$$

Notice that by (2), (3), (6) and (10), there are equilibria  $E^- = (0, T^b, S_w^-)$ and  $E^+ = (\widetilde{Q}^+, T^0, 1)$ .

*Proof.* (1) This was assumed in (46).

(2) This follows from (47) and substitution of  $T = T^{-} = T^{b}$  into Eq. (43).

(3)  $\widetilde{Q}^{+}A(T^{0}) + B(T^{0}) = \widetilde{Q}^{+}A(T^{+}) + B(T^{+}) = 0$  by Eq. (31).

(4) This follows from properties (2) and (3).

(5)  $C(T, S_w) < 0$  because in Eq. (44),  $\rho_w - \rho_g$  and  $\frac{1}{\rho_w} - \frac{1}{\rho_g}$  have opposite sign.

(6)  $D(T^b, S^-_w) = 0$  follows from substituting  $T = T^- = T^b$  into Eq. (45) and using both  $S_w + S_g = 1$  and  $f_w + f_g = 1$ . (7) Differentiating Eq. (45) with respect to  $S_w$ , we obtain, with  $D = D(T, C_w)$ 

 $D(T, S_w),$ 

$$(\rho_w - \rho_g)\frac{\partial \tilde{\Omega}}{\partial S_w}D + (\rho_w - \rho_g)\tilde{\Omega}\frac{\partial D}{\partial S_w} = -\tilde{v}^{SCF}(\rho_w - \rho_g) +$$

$$+ (\widetilde{v}^{SCF} + \frac{\rho_w^-}{\rho_w} \left( -\widetilde{v}^{SCF} S_w^- + f_w^- \right) +$$

$$+\frac{\rho_g^-}{\rho_g}\left(-\widetilde{v}^{SCF}S_g^- + f_g^-\right)\right)\left(\rho_w - \rho_g\right)\frac{\partial f_w}{\partial S_w}.$$
(52)

Hence using property (6) we obtain

$$\left((\rho_w - \rho_g)\widetilde{\Omega}\frac{\partial D}{\partial S_w}\right)^- = \left(\left(\frac{\partial f_w}{\partial S_w}\right)^- - \widetilde{v}^{SCF}\right)(\rho_w^- - \rho_g^-) = 0.$$
(53)

The second equality in Eq. (53) follows from property I of the SCF (Eq. (48)).

(8) The second derivative of Eq. (45) relative to  $S_w$  can be obtained from Eq. (52) as

$$(\rho_w - \rho_g) \frac{\partial^2 \widetilde{\Omega}}{\partial S_w^2} D + 2(\rho_w - \rho_g) \frac{\partial \widetilde{\Omega}}{\partial S_w} \frac{\partial D}{\partial S_w} + (\rho_w - \rho_g) \widetilde{\Omega} \frac{\partial^2 D}{\partial S_w^2} = (54)$$
$$\left( \widetilde{v}^{SCF} + \frac{\rho_w^-}{\rho_w} \left( -\widetilde{v}^{SCF} S_w^- + f_w^- \right) + \frac{\rho_g^-}{\rho_g} \left( -\widetilde{v}^{SCF} S_g^- + f_g^- \right) \right) (\rho_w - \rho_g) \frac{\partial^2 f_w}{\partial S_w^2}.$$

At  $E^-$  the first term and the second term on the left are zero because  $(\partial D/\partial S_w)^- = D^- = 0$ . Thus Eq. (54) at  $E^-$  can be simplified to

$$\left((\rho_w - \rho_g)\widetilde{\Omega}\frac{\partial^2 D}{\partial S_w^2}\right)^- = \left((\rho_w - \rho_g)\frac{\partial^2 f_w}{\partial S_w^2}\right)^- > 0.$$
(55)

The inequality in Eq. (55) follows from Eq. (49).

(9) Substitution of  $T = T^{-}$  in Eq. (45) leads to

$$(\rho_w^- - \rho_g^-) \widetilde{\Omega} (T^-, S_w) D(T^-, S_w) = -\widetilde{v}^{SCF} (\rho_w^- S_w + \rho_g^- S_g) + + (\rho_w^- f_w + \rho_g^- f_g) + \widetilde{v}^{SCF} (\rho_w^- S_w^- + \rho_g^- S_g^-) - (\rho_w^- f_w^- + \rho_g^- f_g^-).$$
 (56)

Using  $S_g = 1 - S_w$ ,  $f_g = 1 - f_w$  this can be rewritten as

$$(\rho_w^- - \rho_g^-) \widetilde{\Omega} (T^-, S_w) D(T^-, S_w) = = (\rho_w^- - \rho_g^-) \left( (f_w - f_w^-) - \widetilde{v}^{SCF} (S_w - S_w^-) \right) > 0$$

The inequality follows from Eq. (50).

(10) This is a tedious but straightforward calculation starting from Eqs. (26), (34) and (27) with  $S_w = 1$  and  $\tilde{Q} = \tilde{Q}^+$ .

(11) The derivative of Eq. (45) relative to T can be written as

$$\left(\widetilde{\Omega}D\frac{d}{dT}\left(\rho_{w}-\rho_{g}\right)+\left(\rho_{w}-\rho_{g}\right)D\frac{d\widetilde{\Omega}}{dT}+\left(\rho_{w}-\rho_{g}\right)\widetilde{\Omega}\frac{dD}{dT}\right)^{-}= (57)$$

$$-\widetilde{v}^{SCF}S_{g}^{-}\left(\frac{d\rho_{g}}{dT}\right)^{-}+\rho_{g}^{-}\left(\frac{d(\rho_{g}^{-1})}{dT}\right)^{-}\left(-\widetilde{v}^{SCF}S_{g}^{-}+f_{g}^{-}\right)\left(\rho_{w}f_{w}+\rho_{g}f_{g}\right)^{-}$$

$$+\left(\widetilde{v}^{SCF}+\frac{\rho_{w}^{-}}{\rho_{w}}\left(-\widetilde{v}^{SCF}S_{w}^{-}+f_{w}^{-}\right)+\frac{\rho_{g}^{-}}{\rho_{g}}\left(-\widetilde{v}^{SCF}S_{g}^{-}+f_{g}^{-}\right)\right)^{-}\left(f_{g}\frac{d\rho_{g}}{dT}\right)^{-}.$$

The first and second term on the left are zero because  $D^- = 0$ , and therefore we obtain

$$\left(\left(\rho_{w}-\rho_{g}\right)\widetilde{\Omega}\frac{dD}{dT}\right)^{-} = -\widetilde{v}^{SCF}S_{g}\left(\frac{d\rho_{g}}{dT}\right)^{-} - \frac{1}{\rho_{g}^{-}}\left(\frac{d\rho_{g}}{dT}\right)^{-}\left(-\widetilde{v}^{SCF}S_{g}^{-}+f_{g}^{-}\right)\left(\rho_{w}f_{w}+\rho_{g}f_{g}\right)^{-} + \left(f_{g}\frac{d\rho_{g}}{dT}\right)^{-},$$
(58)

which can be rearranged to

$$\left(\left(\rho_w - \rho_g\right) \widetilde{\Omega} \frac{dD}{dT}\right)^- =$$

$$= \left(f_g - \widetilde{v}^{SCF} S_g\right)^- \left(\rho_g^- - \left(\rho_w f_w + \rho_g f_g\right)^-\right) \frac{1}{\rho_g^-} \left(\frac{d\rho_g}{dT}\right)^-.$$
(59)

Using  $f_g^- = 1 - f_w^-$  in the second parenthesis, as well as the law of ideal gases for  $d\rho_g/dT$ , we obtain

$$\left(\widetilde{\Omega}\frac{dD}{dT}\right)^{-} = -\left(f_g - \widetilde{v}^{SCF}S_g\right)^{-} f_w^{-} \left(\frac{1}{\rho_g}\frac{d\rho_g}{dT}\right)^{-} =$$
$$= \frac{1}{T^{-}} \left(f_g - \widetilde{v}^{SCF}S_g\right)^{-} f_w^{-} > 0.$$
(60)

The inequality follows from Eq. (51).

We shall need one more property of the system (39)-(41):

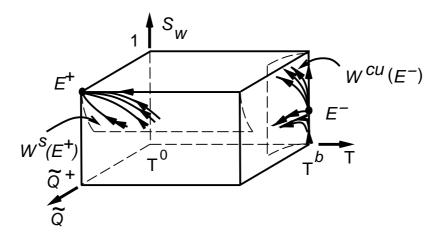
(12)  $\widetilde{q}_b(T-T^b)(S_w-1)C + (\widetilde{Q}A+B)(\widetilde{Q}\frac{\partial C}{\partial T}+\frac{\partial D}{\partial T}) < 0$  at all points  $(\widetilde{Q},T,S_w)$  such that  $0 < \widetilde{Q} \leq \widetilde{Q}^+$ ,  $T_o \leq T < T^b$ ,  $S_w^- < S_w < 1$ , and  $S_w = \widetilde{Q}C(T,S_w) + D(T,S_w) = 0.$ 

In order to justify assuming (12), we make the following observations. The first term in the sum is negative, since C < 0 by (5). The quantity  $\widetilde{Q}A + B$  is nonpositive by property (4). If we solve for  $\widetilde{Q}$  on the surface where  $\widetilde{Q}C + D = 0$  and substitute into  $\widetilde{Q}\frac{\partial C}{\partial T} + \frac{\partial D}{\partial T}$ , we see that it suffices to verify the inequality  $D\frac{\partial C}{\partial T} - C\frac{\partial D}{\partial T} > 0$ . Using the formulas and values of physical quantities given in Appendix A, we have plotted  $D\frac{\partial C}{\partial T} - C\frac{\partial D}{\partial T}$  as a function of T and  $S_w$ , and verified numerically that it is always positive.

#### 5. INVARIANT MANIFOLDS

In the following section, we shall see that (i) at the equilibrium  $E^-$  there is a one-dimensional stable manifold and a two-dimensional centerunstable manifold, and (ii) at the equilibrium  $E^+$  there is a two-dimensional

stable or center-stable manifold and a one-dimensional unstable manifold. Because the center-unstable manifold of  $E^-$  and the stable or center-stable manifold of  $E^+$  are both two-dimensional, it is reasonable to expect that they intersect. The intersection would be a connection from  $E^-$  to  $E^+$ that represents the traveling wave we seek. See Fig. 1.



**FIG. 1.** Invariant manifolds when  $E^+$  has a two-dimensional stable manifold.

## 6. EXISTENCE OF A CONNECTION

To simplify the notation in this section, in Eqs. (39)–(41) we replace  $\widetilde{Q}$  by Q and  $S_w$  by S, and we remove the tildes and w subscripts from related terms. We shall restrict our attention to the set

$$R = \{ (Q, T, S) : 0 \le Q \le Q^+, \ T^0 \le T \le T^b, \ 0 \le S \le 1 \}.$$

Thus we consider the system

$$\dot{Q} = q_b (T - T^b) (S - 1),$$
(61)

$$\dot{T} = QA(T) + B(T), \tag{62}$$

$$\dot{S} = QC(T,S) + D(T,S) \tag{63}$$

on R. From Theorem 1, there are equilibria at  $E^- = (0, T^b, S^-)$  and  $E^+ = (Q^+, T^0, 1)$ .

THEOREM 2. If the system (61)–(63) satisfies property (12) of Section 4, then there is a connection in R from  $E^-$  to  $E^+$ .

*Proof.* We shall frequently use properties (1)-(11) of Theorem 1.

Using property (7), we see that the linearization of Eqs. (61)–(63) at  $E^-$  has the matrix

$$\begin{pmatrix} 0 & q_b(S^- - 1) & 0 \\ A(T^b) & B'(T^b) & 0 \\ C(T^b, S^-) & \frac{\partial D}{\partial T}(T^b, S^-) & 0 \end{pmatrix}.$$
 (64)

The characteristic equation is

$$\lambda \left( \lambda^2 - B'(T^b)\lambda - q_b(S^- - 1)A(T^b) \right) = 0.$$

Thus there is a 0 eigenvalue with eigenvector (0, 0, 1). By (1), because  $q_b(S^- - 1)A(T^b) > 0$ , there are also a negative eigenvalue and a positive eigenvalue. Let  $W^c(E^-)$ ,  $W^u(E^-)$  and  $W^{cu}(E^-)$  denote the one-dimensional center, one-dimensional unstable, and two-dimensional center-unstable manifolds of  $E^-$  respectively.

In R:

1.  $\dot{Q} > 0$  unless  $T = T^b$  or S = 1. There  $\dot{Q} = 0$ .

2.  $\dot{T}$  depends only on Q and T. It has the sign of QA(T) + B(T).

3.  $\dot{S}$  has the sign of QC(T, S) + D(T, S).

See Figures 2 and 3. For i, j = +, -, define

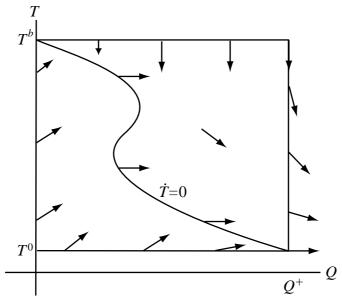
$$R_j^i = \{(Q, T, S) \in \operatorname{Int} R : T \text{ has sign } i, S \text{ has sign } j\}.$$

By Lemma 5.2 below, at  $E^-$  an eigenvector v for the positive eigenvalue points into  $R_-^-$ . Let  $\Gamma$  denote the branch of  $W^u(E^-)$  that is tangent to v. Then near  $E^-$ ,  $\Gamma$  lies in  $R_-^-$ .

The line Q = 0,  $T = T^b$  is invariant under Eqs. (61)–(63) and can be taken to be  $W^c(E^-)$ . Thus  $W^{cu}(E^-)$ , which is tangent at  $E^-$  to the plane spanned by (0, 0, 1) and v, includes this line and  $\Gamma$ . From properties (7) and (8), all solutions in  $R \cap W^{cu}(E^-)$  lie in the sector of  $W^{cu}(E^-)$  between  $\Gamma$  and the portion of the line that is above  $E^-$ , which we denote L. See Figure 4.

Let us parameterize the solution curves in this sector of  $W^{cu}(E^-)$  as  $\Gamma(\theta), 0 \le \theta \le 1$ , with  $\Gamma(0) = \Gamma$  and  $\Gamma(1) = L$ . For  $0 < \theta < 1$ , as  $t \to -\infty$ ,  $\Gamma(\theta)$  approaches  $E^-$  tangent to L to infinite order. Thus all  $\Gamma(\theta)$  with  $0 < \theta < 1$  lie in  $R^-_+$  for large negative t. See Figure 4.

In the interior of R,  $\dot{Q} > 0$ . Thus, as t increases, all  $\Gamma(\theta)$  with  $0 < \theta < 1$  either leave  $R_{+}^{-}$  through its boundary or approach an invariant set contained in its boundary.

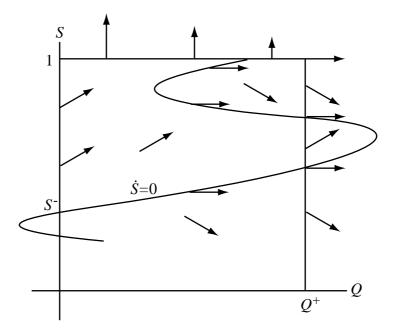


**FIG. 2.** The vector field for fixed  $S, 0 \leq S < 1$ , projected onto the QT-plane. Notice that  $\dot{Q} > 0$  for  $T^0 \leq T < T^b$ ; and  $\dot{T} = 0$  on the curve Q(T) = -B(T)/A(T), a smooth curve with  $Q(T^0) = Q^+, Q(T^b) = 0$ , and  $0 < Q(T) < Q^+$  for  $T^0 < T < T^b$  (a consequence of properties (1)–(4)). Also notice that by property (1),  $\dot{T} < 0$  to the right of this curve and  $\dot{T} > 0$  to the left of this curve. For S = 1 the picture is different, since  $\dot{Q} = 0$ .

The boundary of  $R_+^-$  is contained in the union of the eight sets Q = 0,  $Q = Q^+$ ,  $T = T^0$ ,  $T = T^b$ , S = 0, S = 1,  $\dot{T} = 0$ , and  $\dot{S} = 0$ .

Since (i)  $E^- = (0, T^b, S^-)$ , (ii) all  $\Gamma(\theta)$  with  $0 < \theta < 1$  approach  $E^-$  as  $t \to -\infty$ , and (iii) all have  $\dot{Q} > 0$ ,  $\dot{T} < 0$ , and  $\dot{S} > 0$  while they are in the interior of  $R^-_+$ , we conclude that they cannot leave  $R^-_+$  through Q = 0,  $T = T^b$ , or S = 0. Also, properties (1) and (3) imply that the portion of the set  $T = T^0$  with  $Q < Q^+$  has  $\dot{T} > 0$  and hence does not contain any boundary points of  $R^-_+$ . In addition, no solution can leave  $R^-_+$  through the portion of the set  $\dot{T} = 0$  in the interior of R, because of the direction of the vectors on this set. See Figure 2.

We conclude that, as t increases, all  $\Gamma(\theta)$  with  $0 < \theta < 1$  leave  $R_+^$ through the union of the sets  $Q = Q^+$ ,  $\dot{S} = 0$ , and S = 1. Now no solution can leave  $R_+^-$  through the line segment  $Q = Q^+$ ,  $T = T^0$ ,  $0 \le S < 1$ because on this set  $\dot{Q} > 0$  and  $\dot{T} = 0$ . See the point  $(Q^+, T^0)$  in Figure 2. Also, because the S-coordinate of  $E^-$  is  $S^-$ , no solution  $\Gamma(\theta)$  with  $0 < \theta < 1$  can leave  $R_+^-$  through a point  $(\theta, T, S)$  with  $S \le S^-$ . Combining this information with that in the previous paragraph, we have the following



**FIG. 3.** The vector field for fixed  $T, T^0 \leq T < T^b$ , projected onto the QS-plane. Notice that  $\dot{Q} > 0$  for  $0 \leq S < 1$ , and  $\dot{S} = 0$  on the curve  $Q_T(S) = -D(T,S)/C(T,S)$ , which by property (5) is smooth at least for  $S^- - \epsilon \leq S \leq 1$ . By properties (6)–(9), for  $T = T^b$ , this curve has Q > 0 except at  $S = S^-$ , where it has second-order contact with the S-axis. However, little can be said about it for other values of T. Also notice that by property (5),  $\dot{S} < 0$  to the right of this curve and  $\dot{S} > 0$  to the left of this curve. For  $T = T^{b}$  the picture is different, since Q = 0.

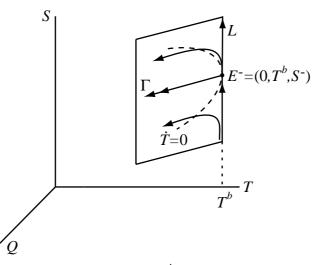
picture. For  $T^0 < T < T^b$  and  $S^- < S < 1$ , define

$$\tilde{Q}(T,S) = \min\left(-\frac{D(T,S)}{C(T,S)}, Q^+\right).$$

Let  $U = \{(T, S) : T^0 < T < T^b, S^- < S < 1, \text{ and } \tilde{Q}(T, S) > 0\}$ , an open subset of TS-space. Then all  $\Gamma(\theta)$  with  $0 < \theta < 1$  leave the interior of  $R_+^$ through one of the following three disjoint sets:

- 1.  $G_1 = \{(Q, T, S) : (T, S) \in U \text{ and } Q = \tilde{Q}(T, S)\},\$
- 2.  $G_2 = \{(Q, T, S) : 0 < Q < Q^+, T^0 < T < T^b, S = 1\},\$ 3.  $G_3 = \{(Q, T, S) : Q = Q^+, T^0 \le T < T^b, S = 1\}.$

The first two sets are two-dimensional.  $G_3$  is a one-dimensional set in the common boundary of  $G_1$  and  $G_2$ .



**FIG. 4.** Part of  $W^{cu}(E^-)$ . The set  $\dot{S} = 0$  meets  $W^{cu}(E^-)$  in a curve, shown dashed, which has second order contact with the line Q = 0,  $T = T^b$  at  $E^-$ . Near  $E^-$ ,  $\Gamma$  lies in the intersection of  $W^{cu}(E^-)$  and  $R^-_-$ . Also, near  $E^-$ , the portion of  $W^{cu}(E^-)$  in  $\operatorname{Int} R$  that is outside the dashed curve is in  $R^+_+$ .

Let  $I_j$  denote the set of  $\theta$  in  $0 < \theta < 1$  such that  $\Gamma(\theta)$  leaves the interior of  $R_+^-$  through  $G_j$ . On  $G_2$ ,  $\dot{S} > 0$ . On  $G_1$ , where  $Q = Q^+$ ,  $\dot{Q} > 0$ ; and where  $Q = -\frac{D(T,S)}{C(T,S)}$ ,  $\dot{S} = 0$ , so by the assumption of the theorem,

$$\nabla \dot{S} \cdot (\dot{Q}, \dot{T}, \dot{S}) = \left(C, Q \frac{\partial C}{\partial T} + \frac{\partial D}{\partial T}, Q \frac{\partial C}{\partial S} + \frac{\partial D}{\partial S}\right) \cdot (\dot{Q}, \dot{T}, 0) =$$
$$= q_b (S - 1)(T - T^b)C + (QA + B) \left(Q \frac{\partial C}{\partial T} + \frac{\partial D}{\partial T}\right)$$

is negative. It follows easily that  $I_j$  is open for j = 1, 2. The sets  $I_1$  and  $I_2$ are disjoint and nonempty:  $I_1$  includes  $\theta$  near 0, and because of property (9),  $I_2$  includes  $\theta$  near 1. Since the interval  $0 < \theta < 1$  is connected, it cannot be the union of disjoint open sets. Therefore there exists  $\theta_0$  in this interval such that  $\Gamma(\theta_0)$  leaves the interior of  $R_+^-$  through  $G_3$  or is asymptotic to an invariant set in  $G_3$ .

On  $G_3$ ,  $\dot{Q} = \dot{S} = 0$ . In fact,  $G_3$  consists of (1) the equilibrium  $E^+$  and (2) the open line segment  $Q = Q^+, T^0 < T < T^b, S = 1$ , on which  $\dot{T} < 0$ by property (4). This segment is a proper subset of a solution curve. Thus  $\Gamma(\theta_0)$  cannot leave the interior of  $R^-_+$  through  $G_3$ , and the only invariant subset of  $G_3$  to which it can be asymptotic is  $E^+$ . We now state and prove Lemma 5.2. To simplify the notation, let

$$q = q_b(S^- - 1) < 0, \quad a = A(T^b) < 0, \quad b = B'(T^b),$$
  
$$c = C(T^b, S^-) < 0, \quad d = \frac{\partial D}{\partial T}(T^b, S^-) > 0, \quad \alpha = \sqrt{b^2 + 4aq}.$$

**Lemma 5.2** The linearization of Eqs. (61)–(63) at  $E^-$  has an eigenvector v for the positive eigenvalue that points into  $R_-^-$ .

*Proof.* With the above notation, Eq. (64) becomes the matrix

$$\begin{pmatrix} 0 & q & 0 \\ a & b & 0 \\ c & d & 0 \end{pmatrix}.$$

The positive eigenvalue is

$$\frac{b+\alpha}{2} > 0. \tag{65}$$

An eigenvector is

$$v = \left(\frac{b+\alpha}{2}, \frac{b^2+\alpha b+2aq}{2q}, *\right),$$

where the last component is not important. Now the expression  $b^2 + \alpha b + 2aq$ is positive for b = 0, and the equation  $b^2 + \alpha b + 2aq = 0$  implies, after a little algebra, that  $a^2q^2 = 0$ . Since this is false, we have

$$b^2 + \alpha b + 2aq > 0 \text{ for all } b. \tag{66}$$

From Eqs. (65) and (66), the first two components of v have signs (+, -). Therefore the vector v, based at  $E^- = (0, T^b, S^-)$ , points into the interior of R.

To see that v points into the interior of  $R_{-}^{-}$ , we regard  $\dot{T}$  and  $\dot{S}$  as functions of (Q, T, S), and we check that at  $E^{-}$ ,  $\nabla \dot{T} \cdot v < 0$  and  $\nabla \dot{S} \cdot v < 0$ . At  $E^{-}$ ,

$$\nabla \dot{T} \cdot v = (a, b, 0) \cdot \left(\frac{b+\alpha}{2}, \frac{b^2 + \alpha b + 2aq}{2q}, *\right) = \frac{1}{2q} (3abq + a\alpha q + b^3 + \alpha b^2).$$

For b = 0, this expression is negative. Setting this expression equal to 0 implies, after some algebra, that  $a^3q^3 = 0$ . Since this is false,  $\nabla \dot{T} \cdot v < 0$  for all b.

At  $E^-$ ,

$$\nabla \dot{S} \cdot v = (c, d, 0) \cdot \left(\frac{b+\alpha}{2}, \frac{b^2+\alpha b+2aq}{2q}, *\right) = \frac{c(b+\alpha)q + d(b^2+\alpha b+2aq)}{2q}$$

From c < 0, q < 0, d > 0, Eqs. (65) and (66), this expression is negative.

The linearization of Eqs. (61)–(63) at  $E^+$  has the matrix

$$\begin{pmatrix} 0 & 0 & q_b(T^0 - T^b) \\ A(T^0) & Q^+ A'(T^0) + B'(T^0) & 0 \\ C(T^0, 1) & 0 & Q^+ \frac{\partial C}{\partial S}(T^0, 1) + \frac{\partial D}{\partial S}(T^0, 1) \end{pmatrix}.$$

The characteristic equation is

$$0 = \left(\lambda - Q^+ A'(T^0) - B'(T^0)\right) \cdot \left(\lambda^2 - \left(Q^+ \frac{\partial C}{\partial S}(T^0, 1) + \frac{\partial D}{\partial S}(T^0, 1)\right) \lambda - q_b(T^0 - T^b)C(T^0, 1)\right).$$

The eigenvalues are  $Q^+A'(T^0) - B'(T^0)$  and two whose product is  $-q_b(T^0 - T^b)C(T^0, 1)$ . Assumptions (3) and (4) imply that  $Q^+A'(T^0) - B'(T^0) \leq 0$ . Since  $q_b(T^0 - T^b)C(T^0, 1) > 0$ , the other two eigenvalues are one positive and one negative. Thus at  $E^+$  there is a one-dimensional stable manifold and a two-dimensional stable or center-stable manifold. This fact was used in Section 5.

Acknowledgements: We thank C. J. van Duijn for many useful discussions and suggestions. We also thank Shell for financial support of our research on steamdrive recovery of oil and pollutant products. We also thank Beata Gundelach for careful and expert typesetting of this paper. We want to thank the anonymous referee for many useful suggestions.

#### REFERENCES

- AYDELOTTE, S.R., AND POPE, G.A., A simplified predictive model for steamdrive performance, J. PET. TECHN. (MAY 1983), 991–1002.
- 2. BEAR, J., Dynamics of Fluids in Porous Media, DOVER, NEW YORK (1972), 650-651.
- 3. BIRD, R.B., STEWART, W.E., AND LIGHTFOOT, E.N., *Transport Phenomena*, WILEY, NEW YORK (1960).
- 4. BROOKS, R.H., AND COREY, A.T., Properties of porous media affecting fluid flow, J. IRR. DRAIN. DIV., PROC., AM. SOC. CIV. ENG. (JUNE 1966), 61–88.
- BRUINING, J., MARCHESIN, D., AND DUIJN, C.J., Steam injection into water-saturated porous rock, COMPUTATIONAL AND APPLIED MATHEMATICS, IN PRESS (2004), AND IMPA PREPRINT NO. 136/2002 (2002).
- 6. DAKE, L.P., Fundamentals of Reservoir Engineering, ELSEVIER, AMSTERDAM (1978).
- DULLIEN, F.A.L., Porous Media: Fluid Transport and Pore Structure, ACADEMIC PRESS, New YORK (1979).

- HUNT, J.R., SITAR, N, AND UDELL, K.S., Non-aqueous phase liquid transport and clean up: Part I, analysis of mechanisms. WATER RESOURCES RESEARCH 24 (1988), 1247– 1258.
- HUNT, J.R., SITAR, N, AND UDELL, K.S., Non-aqueous phase liquid transport and clean up: Part II, experimental studies, WATER RESOURCES RESEARCH 24 (1988), 1259– 1269.
- 10. LEVERETT, M.C., Capillary behavior in porous solids, TRANS., AIME 142 (1941), 142–169.
- 11. MANDL, G.W., AND VOLEK, C.W., Heat and mass transport in steamdrive processes, SPEJ (MARCH 1969), 57–79.
- 12. MARX, J.W., AND LANGENHEIM, R.H., Reservoir heating by hot fluid injection, TRANS., AIME **216** (1959), 312.
- MENEGUS, D.K., AND UDELL, K.S., A study of steam injection into water saturated capillary porous media, HEAT TRANSFER IN POROUS MEDIA AND PARTICULATE FLOWS, ASME HEAT TRANSFER DIV., NEW YORK 46 (1985), 151–157.
- OLEĬNIK, O., Discontinuous Solutions of Nonlinear Differential Equations, USP. MAT. NAUK. (N. S.) 12, (1957), 3–73. ENGLISH TRANSL. IN AMER. MATH SOC. TRANSL. SER. 2, 26, 95–172.
- 15. PETERSON, P.F., Diffusion layer modeling for condensation with multicomponent non-condensable gases, JOURNAL OF HEAT TRANSFER, TRANSACTIONS OF THE ASME **122** (2000), 716-720.
- 16. PRATS, M., *Thermal Recovery*, MONOGRAPH SERIES, SPE, RICHARDSON, TEXAS 7 (1982).
- SHODA, A., WANG, C.Y., AND CHENG, P., Simulation of constant pressure steam injection in a porous medium, INT. COMM. HEAT MASS TRANSFER 25 (1998), 753–762.
- SHUTLER, N.D., A one-dimensional analytical technique for predicting oil recovery by steamflooding, SPEJ (DECEMBER 1972), 489–498.
- 19. STEWART, L.D., AND UDELL, K.S. Mechanisms of residual oil displacement by steam injection, SPE Reservoir Engineering (November 1988), 1233–1242.
- 20. TAMIM, M., ABOU-KASSEM, J.H., AND FAROUQ ALI, S.M., Recent developments in numerical simulation techniques of thermal recovery processes, JOURNAL OF PETROLEUM ENGINEERING SCIENCE AND ENGINEERING 26 (2000), 283–289.
- 21. TORTIKE, W.S., AND FAROUQ ALI, S.M., Saturated-steam-property functional correlations for fully implicit reservoir simulation, SPERE (NOVEMBER 1989), 471–474.
- 22. UDELL, K.S., Heat transfer in porous media considering phase change and capillarity—the heat pipe effect, INTL. J. HEAT MASS TRANSFER 28 (1985), 485–495.
- WEAST, R.C., CRC Handbook of Chemistry and Physics, 58<sup>th</sup> EDITION, CRC PRESS, CLEVELAND, OHIO (1977–1978).
- WINGARD, J.S., AND ORR, F.M., An analytical solution for steam/oil/water displacements, SPE Advanced Technology Series 2 (1994), 167–176.
- YORTSOS, Y.C., Distribution of fluid phases within the steam zone in steam-injection processes, SPEJ (AUGUST 1984), 458–466.
- 26. YORTSOS, Y.C., AND GAVALAS, G.R., Heat transfer ahead of moving condensation fronts in thermal recovery processes, INT. J. HEAT AND MASS TRANSFER 25 (1982), 305–316.

# APPENDIX: PHYSICAL QUANTITIES, SYMBOLS AND VALUES

In this Appendix we summarize the values and units of the various physical quantities used in the computation and empirical expressions for the various parameter functions. For convenience we express the heat capacity of the rock  $C_r^p$  in terms of energy per unit volume of *porous medium* per unit temperature, *i.e.*, the factor  $1 - \varphi$  is included in the rock density. All other densities are expressed in terms of mass per unit volume of the phase.

#### Temperature-dependent properties of steam and water

We use reference [21] to obtain all the temperature-dependent properties below. The water and steam densities used to obtain the enthalpies are defined at the bottom. First we obtain the boiling point  $T^b$  at the given pressure p, *i.e.* 

$$T^{b} = 280.034 + \ell \Big( 14.0856 + \ell \Big( 1.38075 + \ell (-0.101806 + 0.019017\ell) \Big) \Big),$$
(A.1)

where  $\ell = \log(p)$  and p is the pressure in [k Pa]. The evaporation heat [J/kg] is given as a function of the temperature T at which the evaporation occurs. We use atmospheric pressure (p = 101.325 [k Pa]) in our computations, to make the example representative of subsurface contaminant cleaning.

The liquid water enthalpy  $h_w(T)$  [J/kg] as a function of temperature is approximated by

$$h_w(T) = 2.36652 \times 10^7 - 3.66232 \times 10^5 T + 2.26952 \times 10^3 T^2 - - 7.30365T^3 + 1.30241 \times 10^{-2}T^4 - - 1.22103 \times 10^{-5}T^5 + 4.70878 \times 10^{-9}T^6.$$
 (A.2)

The steam enthalpy  $h_g~[{\rm J/kg}]$  as a function of temperature is approximated by

$$h_g(T) = -2.20269 \times 10^7 + 3.65317 \times 10^5 T - 2.25837 \times 10^3 T^2 + + 7.3742T^3 - 1.33437 \times 10^{-2}T^4 + 1.26913 \times 10^{-5}T^5 - - 4.9688 \times 10^{-9}T^6.$$
(A.3)

For the latent heat  $h_q^l$  [J/kg] or evaporation heat  $\Lambda(T)$  we obtain

$$h_g^l(T) = \left(7.1845 \times 10^{12} + 1.10486 \times 10^{10}T - 8.8405 \times 10^7 T^2 + 1.6256 \times 10^5 T^3 - 121.377 T^4\right)^{\frac{1}{2}}.$$
(A.4)

The sensible heat of steam  $H^s_g(T)$  in  $[{\rm J}/{\rm m}^3]$  is given as

$$H_g^s(T) = \rho_g \left( h_g(T) - h_w(T^0) - \Lambda(T^0) \right).$$
(A.5)

We also use the temperature-dependent steam viscosity

$$\mu_g(T) = -5.46807 \times 10^{-4} + 6.89490 \times 10^{-6}T - 3.39999 \times 10^{-8}T^2 + 8.29842 \times 10^{-11}T^3 - 9.97060 \times 10^{-14}T^4 + 4.71914 \times 10^{-17}T^5.$$
(A.6)

The temperature-dependent water viscosity  $\mu_w$  is approximated by

$$\mu_w(T) = -0.0123274 + \frac{27.1038}{T} - \frac{23527.5}{T^2} + \frac{1.01425 \times 10^7}{T^3} - \frac{2.17342 \times 10^9}{T^4} + \frac{1.86935 \times 10^{11}}{T^5}.$$
(A.7)

For the steam density as a function of temperature T[K] we use a different expression than [21] because our interest is a steam density at constant pressure, which is not necessarily in equilibrium with liquid water. We use:

$$\rho_g(T) = p \frac{M_{H_2O}}{ZRT} \tag{A.8}$$

where p is the pressure, R=8.31 [J/mol K] and Z is the Z-factor (see e.g. [6]). For the atmospheric pressures of interest here the Z-factor is close to unity so we use Z = 1. The liquid water density as a function of the temperature T[K] is given as

$$\rho_w(T) = 3786.31 - 37.2487T + 0.196246T^2 - 5.04708 \times 10^{-4}T^3 + + 6.29368 \times 10^{-7}T^4 - 3.08480 \times 10^{-10}T^5.$$
(A.9)

	Table 1. Summary of physical parameters and variables			
Physical quantity	Symbol	Value	Unit	
Water, steam fractional functions	$f_w, f_g$	Eq. (11).	$[m^{3}/m^{3}]$	
Porous rock permeability	k	$1.0 \times 10^{-12}$	$[m^3]$	
Water, steam relative permeabilities	$k_{rw}, k_{rg}$	Eqs. (A.10), (A.11).	$[m^3/m^3]$	
Pressure	p	$1.0135 \times 10^{5}$	[Pa]	
Mass condensation rate	q	Eq. (5).	$\left[ \text{kg} / (\text{m}^3 \text{s}) \right]$	
Mass condensation rate coefficient	$q_b$	0.01	$[\text{kg}/(\text{m}^3\text{sK})]$	
Water, steam phase velocity	$u_w, u_g$	Eq. (4).	$[m^3/(m^2s)]$	
Total Darcy velocity		$u_w + u_g$ , Eq. (15).	$[m^3/(m^2s)]$	
SCF velocity	$v^{SCF}$	Eq. (A.16).	[m/s]	
Effective rock heat capacity	$C_r^p$	$2.029 \times 10^{6}$	$\left[ J/(m^3 K) \right]$	
Steam enthalpy per unit mass	$h_{g}$	Eq. (A.3)	[J/kg]	
Rock enthalpy per unit mass	$h_r^g$	$C_{r}^{p}(T-T^{0})/\rho_{r}$	[J/kg]	
Water enthalpy per unit mass	$h_w$	(A.2)	[J/kg]	
Steam enthalpy per unit volume	$H_{a}$	$\rho_q(T)h_q(T)$	$[J/m^3]$	
Steam sensible heat per unit volume	$H_{a}^{g}$	Eq. $(A.5)$	$\left[ J/m^{3} \right]$	
Steam latent heat per unit volume	$H^{\eta}_{L}$	$\rho_q(T)\bar{\Lambda}$	$\left[\mathrm{J/m^{3}}\right]$	
Rock enthalpy per unit volume	$\begin{vmatrix} H_g^s \\ H_g^l \\ H_g^l \\ H_r \end{vmatrix}$	$C_r^p(T-T^0)$	$\left[ J/m^{3} \right]$	
Water enthalpy per unit volume	$H_w$	$\rho_w(T)h_w(T)$	$[J/m^3]$	
Steam total enthalpy at boil. temp.	$H_g^{\widetilde{B}}$	Eq. (A.17).	$\left[ J/m^{3} \right]$	
Capillary pressure	$P_c^{g}$	Eq. (A.12).	[Pa]	
Condensation rate parameter	$q_b$	Eq. (5)	$[\text{kg}/\text{m}^3/\text{K}]$	
Cumulative mass condensation	Q	Eq. (19).	$[\text{kg /m}^3]$	
Water, steam saturations	$\tilde{S}_w, S_g$	Dependent variables.	$[m^3/m^3]$	
Connate water saturation	$S_{wc}$	0.15	$[m^{3}/m^{3}]$	
Temperature		Dependent variable.	[ [K]	
Reservoir temperature	$T^0$	293.	[K]	
Boiling point of water–steam	$T^b$	Eq. (A.1).	[K]	
Water, steam thermal conductivity	$\kappa_w, \kappa_a$	0.652, 0.0208	[W/(mK)]	
Rock, composite thermal conductivity	$\kappa_r, \kappa$	1.83, Eq. (9).	[W/(mK)]	
Water, steam viscosity	$\mu_w, \mu_g$	Eq. (A.7), Eq. (A.6).	[Pa s]	
Rock density	$\rho_s$	2650	$[kg/m^3]$	
Effective rock density	$\rho_r$	$(1-\varphi)\rho_s;$	$[kg/m^3]$	
Water, steam densities	$\rho_w, \rho_g$	Eq. (A.9), Eq. (A.8).	$[kg/m^3]$	
Interfacial tension	$\sigma_{wg}$	$58 \times 10^{-3}$	[N/m]	
Rock porosity		0.38	$[m^{3}/m^{3}]$	
Water evaporation heat	$\left  \begin{array}{c} \varphi \\ \bar{\Lambda} \end{array} \right $	Eq. (3).	[J/kg]	
Capillary diffusion coefficient	Ω	Eq. (13), Eq. (14).	$[m^{3}/m^{3}]$	

# A.1. CONSTITUTIVE RELATIONS

The relative permeability functions  $k_{rw}$  and  $k_{rg}$  are considered to be power functions of their respective saturations [7], *i.e.*,

$$k_{rw} = \begin{cases} \left(\frac{S_w - S_{wc}}{1 - S_{wc}}\right)^{n_w} & \text{for } S_w \ge S_{wc}, \\ 0 & \text{for } 0 \le S_w \le S_{wc}, \end{cases}$$
(A.10)  
$$k_{rg} = \left(\frac{S_g}{1 - S_{wc}}\right)^{n_g}.$$
(A.11)

For the computations we take  $n_w = 4$ ,  $n_g = 2$ . The connate water saturation  $S_{wc}$  is given in the table.

The capillary pressure is of the Brooks-Corey type. We use the dimensionless capillary pressure  $P_c(S_w = 0.5)/(\sigma_{wg}\sqrt{\varphi/k}) = 0.5$ , which is appropriate for many sandstones. The capillary pressure between steam and water is given by an empirical expression that combines Leverett's approach to non-dimensionalize the capillary pressure [10] with the semi-empirically determined saturation dependence suggested in [4]:

$$P_c = \sigma_{wg} \gamma \sqrt{\frac{\varphi}{k}} \left(\frac{\frac{1}{2} - S_{wc}}{1 - S_{wc}}\right)^{\frac{1}{\lambda_s}} \left(\frac{S_w - S_{wc}}{1 - S_{wc}}\right)^{-\frac{1}{\lambda_s}}, \qquad (A.12)$$

where  $\gamma$  is a parameter that in many cases assumes values between 0.3 and 0.7. We use  $\gamma = 0.5$  and  $\lambda_s = \frac{1}{2}$ . Finally,  $\sigma_{wg} = 0.058$  N/m is the water-vapor interfacial tension. We disregard its temperature dependence and use the value at the boiling point (see [23], p. F-45).

### A.2. STEAM CONDENSATION FRONT VELOCITY

Equating  $Q^+$  from Eqs. (29), (30), we obtain:

$$(\rho_w f_w + \rho_g f_g)^- - \tilde{v}^{SCF} \left(\rho_w S_w + \rho_g S_g\right)^- = \rho_w^+ \left(\tilde{u}^+ - \tilde{v}^{SCF}\right).$$
(A.13)

Similarly, we obtain from a Hugoniot condition derived from (8):

$$(H_w f_w + H_g f_g)^- - \widetilde{v}^{SCF} (H_r + \varphi H_w S_w + \varphi H_g S_g)^- =$$
  
=  $\widetilde{u}^+ H_w^+ - \widetilde{v}^{SCF} (H_r^+ + \varphi H_w^+) = 0.$  (A.14)

The right-hand side of Eq. (A.14) vanishes because  $T^+ = T^0$ , see Eq. (1). For the *SCF* velocity it follows from Eqs. (A.13), (A.14) that

$$\widetilde{u}^{+} = \left(\frac{\rho_{g}^{-}}{\rho_{w}^{+}}f_{g}^{-} + \frac{\rho_{w}^{-}}{\rho_{w}^{+}}f_{w}^{-}\right) - \widetilde{v}^{SCF}\left(\frac{\rho_{g}^{-}}{\rho_{w}^{+}}S_{g}^{-} + \frac{\rho_{w}^{-}}{\rho_{w}^{+}}S_{w}^{-} - 1\right), \qquad (A.15)$$

$$\frac{\varphi v^{SCF}}{u^{-}} = \tilde{v}^{SCF} = \frac{H_w^- f_w^- + H_g^B f_g^-}{H_r^- + \varphi H_w^- S_w^- + \varphi H_g^B S_g^-},$$
(A.16)

where we use the nomenclature that follows from Eq. (3):

$$H_g^B \equiv H_g(T^-) = H_g^s(T^-) + H_g^l = H_g^{s-} + \rho_g^- \bar{\Lambda}.$$
 (A.17)

From Eqs. (A.15) and (A.16), we can write  $\tilde{u}^+$ :

$$\widetilde{u}^{+} = \left(\frac{\rho_{w}^{-}}{\rho_{w}^{+}}f_{w}^{-} + \frac{\rho_{g}^{-}}{\rho_{w}^{+}}f_{g}^{-}\right) - \left(\frac{\rho_{w}^{-}}{\rho_{w}^{+}}S_{w}^{-} + \frac{\rho_{g}^{-}}{\rho_{w}^{+}}S_{g}^{-} - 1\right)\frac{H_{w}^{-}f_{w}^{-} + H_{g}^{B}f_{g}^{-}}{H_{r}^{-} + \varphi H_{w}^{-}S_{w}^{-} + \varphi H_{g}^{B}S_{g}^{-}}.$$
(A.18)